## UNIVERSITY OF PUNE

[4362]-116
S.E.(Mechanical / Automobile) Examination - 2013
(2008 pattern)
THEORY OF MACHINES -1
[Total No. of Questions: 12]
[Total No. of Printed pages: 7]
[Time: 4 Hours]
[Max. Marks: 100]

## Instructions:

(1) Answer three questions from Section I and three questions from Section II
(2) Answers to the two sections should be written in separate answer-books
(3) Black figures to the right indicate full marks.
(4) Neat diagrams must be drawn wherever necessary.
(5) Use of logarithmic tables, slide rule, Mollier charts, electronic pocket calculator and steam tables is allowed.
(6) Assume suitable data, if necessary.

## SECTION -I

## UNIT- I

Q1. A) Define the Following Term
i) Grubler's criterion
ii) Structure
iii) Degree of Freedom
iv) Mechanism
B) Explain with neat sketch whitworth Quick Return Mechanism.
C) What is a condition of correct steering? With the help of neat sketch explain the construction and working of Davis Steering Gear mechanism.

## OR

Q2. A) Define the following
i) Machine
ii) Kinematic Chain
iii) Kinematic Link
iv] Grashoff's law
B) Explain the following terms with suitable examples.
i] Completely constrained motion
ii] Incompletely constrained motion
iii] Successfully constrained motion
C) Write short note on
i) Pantograph along with its application.
ii) Scotch yoke mechanism

## UNIT-II

Q3. A) Fig (1) Shows mechanism in which crank OA is rotating clockwise at 240 rpm . At the instant Shown, locate all ICRs for the mechanism and find the velocity of slider E as well as the angular velocity of the link BC using ICR method.
$\mathrm{BC}=\mathrm{OC}=\mathrm{DE}=100 \mathrm{~mm}$
$D$ is a midpoint of $B C$.
$\mathrm{OA}=\mathrm{AB}=50 \mathrm{~mm}$.
$\underline{\operatorname{Fig}(1)}$


B] State and Explain Kennedy's Theorem 'of three centers in line.

Q4. A] In the Toggle mechanism shown in $\underline{\text { Fig (2), the slider D is }}$ counter clockwise direction at a speed of 180 r.p.m Increasing at the rate of $50 \mathrm{rad} / \mathrm{s}^{2}$. The dimensions of the various links are as follows: $\mathrm{OA}=180 \mathrm{~mm}$, $\mathrm{CB}=240 \mathrm{~mm}, \mathrm{AB}=360 \mathrm{~mm}$ and $\mathrm{BD}=540 \mathrm{~mm}$.

For the given configuration,
Find i) Velocity of slider D
ii) Angular velocity of $\mathrm{BD}, \mathrm{AB}, \mathrm{CB}$.
iii) Acceleration of slider D
iv) Angular acceleration of BD.


## Fig (2)

## UNIT-III

Q5. A] In a whitworth Quick return motion, as shown in fig (3),
OA is a crank rotating at 30 r.p.m. in a clockwise direction. The dimensions of various links are $\mathrm{OA}=150 \mathrm{~mm}, \mathrm{OC}=100 \mathrm{~mm}, \mathrm{CD}=125 \mathrm{~mm}$ and $\mathrm{DR}=500$ mm.

Determine-
i) The acceleration of the sliding block $R$
ii) The angular acceleration of the slotted lever CA
$\underline{\operatorname{Fig}(3)}$


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## OR

Q6. A) In a slider crank mechanism, the crank is 60 mm long and
connection rod 240 mm long. When the crank has moved through $40^{\circ}$ from the Inner dear centre position, the velocity of slider is $3 \mathrm{~m} / \mathrm{s}$.

Find i) Angular acceleration of connecting rod.
ii) Acceleration of centre of gravity of connecting rod, which is situated at a distance of 120 mm from big end.

Use Klein's construction.
B) Explains the procedure to construct Klein's construction to
determine the acceleration of a slider crank mechanism. Also, find angular acceleration of connecting rod.

## SECTION- II

UNIT- IV
Q7. A) An IC engine runs at 1600 rpm , has length of connecting rod 240
mm and obliquity ratio 4 . Determine at $40 \%$ of outstroke
i] Angular position of the crank
ii] Linear velocity and acceleration of the piston
iii] Angular velocity and Angular acceleration of connecting rod
B) What is loop closure equation? Derive the same for offset slider crank mechanism.

## OR

Q8. A) Two shafts, angle between whose axes is $20^{\circ}$ are
connected by Hook's joint. Find the angle turned through by the driving shaft rotates when:
i) The velocity ratio is maximum, minimum and unity
ii) The retardation of driven shaft is maximum
iii) Draw the polar diagram representing angular velocities of driving and driven shaft indicating the various angular positions.
B) Explain complex number method of acceleration analysis.

## UNIT-V

Q9. A) A four bar mechanism is used to generate the function $y=x^{2}+4 x$,
for the range $1 \leq x \leq 3$. Find the three precision positions from chebychev spacing, if the initial values of the crank angle and follower angle are $30^{\circ}$ and $150^{\circ}$ respectively. Take $\Delta \theta=\Delta \Phi=90^{\circ}$. Find the corresponding values of $\mathrm{x}, \mathrm{y}, \theta$ and $\Phi$. Also find the dimensions of other link, if the grounded link is 100 mm and input link is 40 mm . Use inversion method.

B] Explain
i] Number Synthesis
ii] Type Synthesis
iii] Dimensional Synthesis

## OR

Q10. A) Explain following terms
i) Function Generation
ii) Path Generation
iii) Precision Points
iv) Structural Error
B) Design a four bar mechanism with input link 'a 'and output link ' $c$ ' angles $\theta$ and $\Phi$ for three successive positions are as follows

$$
\begin{array}{ll}
\theta_{1}=20^{0} & \Phi_{1}=35^{0} \\
\theta_{2}=35^{0} & \Phi_{2}=45^{0} \\
\theta_{3}=50^{0} & \Phi_{3}=60^{0}
\end{array}
$$

If the length of grounded link is 40 mm , using Freudenstein's equation find out other link lengths to satisfy the given positional conditions.

## UNIT-VI

Q11 A) With the help of neat sketch explain Bifilar suspension method
B) Explain dynamic equivalence of two mass systems, for a connecting rod of an IC engine having mass ' $m$ ' and radius of gyration ' $k$ '. Obtain a two mass dynamically equivalent system, having one of the two masses at the small end. How dynamical equivalence is achieved if it is required that the other mass located at the big end

## OR

Q12. A) An IC engine has a stroke of 100 mm and bore of 80 mm .
The connecting rod is 160 mm between centers and has total mass of 1.3 kg . Its center of mass is 130 mm away from small end center and radius of gyration about the mass center is 75 mm . The reciprocating mass is 1.8 kg . Determine magnitude of resultant forces on the crank pin, neglecting friction and gravity, when the crank is $30^{\circ}$ after the TDC position and rotating at 1600 rpm clockwise. The gas pressure on the piston is $2 \mathrm{~N} / \mathrm{mm}^{2}$
B) Derive frequency equätion oflompound pendulum

